# **Constrained Space Deformation for Design Optimization**

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Common target designs are sheet metal surfaces



# **Mesh-Based Surface Deformation**

## Thin Shell Deformation <sup>1</sup>

- Physically-inspired technique suitable for sheet metal surfaces
- Flexible modeling of material behavior
- Based on the minimization of stretching and bending energies



<sup>1.</sup> Botsch et al., On Linear Variational Surface Deformation Methods, Trans. on Visualization and Computer Graphics, 2008

#### Thin Shell Deformation

Measure stretching and bending by 1<sup>st</sup> and 2<sup>nd</sup> order partial derivatives of the displacement function  $d: S \to \mathbb{R}^3$ 

$$E_{\text{stretch}}[\boldsymbol{d}] = \int_{\mathcal{D}} \|\nabla \boldsymbol{d}(\boldsymbol{x})\|^2 \, \mathrm{d}\boldsymbol{x}$$
$$E_{\text{bend}}[\boldsymbol{d}] = \int_{\mathcal{D}} \|\Delta \boldsymbol{d}(\boldsymbol{x})\|^2 \, \mathrm{d}\boldsymbol{x}$$
$$E_{\text{fix}}[\boldsymbol{d}] = \int_{\mathcal{H}\cup\mathcal{F}} \|\boldsymbol{d}(\boldsymbol{x}) - \bar{\boldsymbol{d}}(\boldsymbol{x})\|^2 \, \mathrm{d}\boldsymbol{x}$$



#### Surface-Based Discretization

Discretize on the mesh using standard differential operators<sup>2</sup>

$$E_{\text{stretch}}[\boldsymbol{d}_{h}] = \sum_{\boldsymbol{x}_{i} \in \mathcal{D}} A_{i} \|\nabla \boldsymbol{d}_{i}\|^{2}$$
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→ Solve a linear system:

$$\left(w_{s}\boldsymbol{G}^{T}\boldsymbol{G}+w_{b}\boldsymbol{L}^{T}\boldsymbol{L}+w_{f}\boldsymbol{F}^{T}\boldsymbol{F}\right)\boldsymbol{d} = w_{f}\boldsymbol{F}^{T}\boldsymbol{F}\boldsymbol{d}$$

<sup>2.</sup> Meyer et al., Discrete Differential-Geometry Operators for Triangulated 2-Manifolds, Visualization and Mathematics, 2003

#### Limitations

• Assumption: Surface  $\mathcal{S}$  is a proper triangle mesh

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- Instead of deforming the surface  $\mathcal S,$  deform embedding space  $\Omega$ 
  - Disconnected components
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- Use meshless approximation methods
- Represent *d* by basis functions  $\varphi_i$  located at centers  $c_i$ :

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- 3. Lloyd relaxation (k-means clustering)



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$$\boldsymbol{d} = \boldsymbol{\Phi} \boldsymbol{w}$$
 with  $\boldsymbol{\Phi}_{ij} = \varphi_j(\boldsymbol{x}_i)$ 

• Leads to a modified linear system:

$$\boldsymbol{\Phi}^{T}\left(\boldsymbol{w}_{s}\boldsymbol{G}^{T}\boldsymbol{G}+\boldsymbol{w}_{b}\boldsymbol{L}^{T}\boldsymbol{L}+\boldsymbol{w}_{f}\boldsymbol{F}^{T}\boldsymbol{F}\right)\boldsymbol{\Phi}\boldsymbol{w} = \boldsymbol{\Phi}^{T}\left(\boldsymbol{w}_{f}\boldsymbol{F}^{T}\boldsymbol{F}\boldsymbol{\bar{d}}\right)$$

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- Global support  $\rightarrow$  dense linear systems


• Compactly supported RBFs

$$\varphi_j(\mathbf{x}) = \varphi\left(\left\|\mathbf{x} - \mathbf{c}_j\right\|\right) = \varphi(r) = \begin{cases} (1-r)^4(4r+1), & r < \sigma, \\ 0, & \text{otherwise}. \end{cases}$$

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$$\varphi_j(\mathbf{x}) = \mathbf{p}(\mathbf{x})^T \mathbf{M}^{-1}(\mathbf{x}) \mathbf{p}(\mathbf{c}_j) w(\mathbf{x} - \mathbf{c}_j)$$

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• More complex form, inversion of *M* required

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- Replace vertex-based integration over the surface  ${\mathcal S}$  with purely space-based integration method
- Use Lloyd-based sampling to determine integration points  $q_i$



• Evaluate gradients and Laplacians of  $\varphi_i$  at integration points  $q_i$ :

$$\widetilde{E}_{\text{stretch}} = \sum_{i=1}^{N} V_i \|\nabla \boldsymbol{d}(\boldsymbol{q}_i)\|^2 = \sum_{i=1}^{N} V_i \left\|\sum_{j=1}^{k} \boldsymbol{w}_j \nabla \varphi_j(\boldsymbol{q}_i)\right\|^2$$
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## **Constrained Space Deformation**

- Design prototypes contain important geometric features
  - Planar components
  - Circular couplings or wheelhouses
  - Characteristic feature lines



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- Deforming the design during optimization distorts features
  - Impaired functionality
  - Violated production limitations
- · Classical solution: Add penalty terms to the cost function
  - *Creation* of infeasible designs
  - Costly evaluation (e.g., CFD)



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  - + Only create feasible designs
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- Constrained deformation techniques
  - Most methods are surface-based
  - Projection-based constraints<sup>4</sup>

<sup>4.</sup> Bouaziz et al., Shape-Up: Shaping Discrete Geometry with Projections, Computer Graphics Forum, 2012

#### **Projection-Based Constraints**

- Define projection operators *P<sub>c</sub>*: Plane, circle, ...
- · Minimize deviation from prescribed constraints

$$E_{\text{constr}}(\boldsymbol{x}) = \sum_{c=1}^{s} \|\boldsymbol{x} - P_{c}(\boldsymbol{x})\|^{2}$$

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- Projections  $P_c$  typically are nonlinear functions of x
- → Minimize  $E_{\text{constr}}$  by iterative alternating optimization

#### Geometric Constraint Examples

Fundamental geometric constraints: Planarity, circularity, feature lines



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#### Volume Deformation Examples

Comparison to previous results<sup>5</sup>

- Original mesh quality: 0.98
- After RBF deformation: 0.951
- Using our new method: 0.954



<sup>5.</sup> Sieger et al., RBF Morphing Techniques for Simulation-based Design Optimization, Engineering with Computers, 2014.

# Summary & Outlook

- A deformation technique for design optimization
  - + Modeling flexibility like surface-based methods
  - + Representation-independence of space deformations
  - + High quality comparable to RBFs
  - + Improved scalability through sparse linear systems
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  - Precomputation time of MLS basis functions
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- Central idea: Improve the design optimization process by integrating constraints *directly* into the deformation
- Observation: Significant differences in the modeling flexibility and quality of RBFs and MLS
## Future Work

- Additional constraint types:
  - Rigid components
  - Width, height, distances
  - Symmetry relations
  - Angle relations
- Automatic constraint detection
- Performance improvements

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## **Thanks for your attention!**